

TABLE 6-4 Additional Frictional Loss for Turbulent Flow through Fittings and Valves*

Type of fitting or valve	Additional friction loss, equivalent no. of velocity heads, K
45° ell, standard ^{b,c,d}	0.35
45° ell, long radius ^b	0.2
90° ell, standard ^{b,c,d}	0.75
Long radius ^{b,c,d}	0.45
Square or round ^b	1.5
Tee ^b bend, elbow return ^{b,c,d}	1.75
Tee, standard, along run, branch blanked off ^b	0.4
Closed or ell, entering run ^b	1.0
Closed or ell, entering branch ^b	1.0
Branching flow ^b	1 ^e
Coupling ^b	0.04
Union ^b	0.04
Gate valve ^{b,c,d} open	0.17
to open ^b	0.9
to open ^b	4.5
to open ^b	24.0
Diaphragm valve ^b open	2.5
to open ^b	2.5
to open ^b	4.5
to open ^b	25.0
Globe valve ^{b,c}	
Recd seat, open	6.0
to open ^b	9.5
Composition seat, open	6.0
to open ^b	9.5
Plug cock, open	9.0
to open ^b	13.0
to open ^b	26.0
to open ^b	112.0
Angle valve ^b open	2.0
Y or Johnson ^f valve ^{b,c} open	3.0
Plug, cock ^b	
to 15°	0.05
to 15°	0.20
to 20°	1.50
to 40°	17.5
to 90°	206.0
Butterfly valve ^b	
to 15°	0.04
to 15°	0.52
to 20°	1.54
to 40°	10.5
to 90°	118.0
Check valve ^{b,c,d} swing	
Disk	10.0
Ball	20.0
Foot valve ^b	15.0
Water meter ^b disk	7.0
Prism	15.0
Rotary (star-shaped disk)	10.0
Turbine wheel	6.0

*Lapple, *Chem. Eng.*, **56**(7), 96-104 (1949), general survey reference.
^b"Loss of Fluids through Valves, Fittings, and Pipes," Tech. Pap. 439, Crane Co., 1929.
^cForrester, *Experiments upon the Flow of Water in Pipes and Pipe Fittings*, American Society of Mechanical Engineers, New York, 1911.
^dGreninger, *J. Am. Soc. Heat Vent Eng.*, **20**, 493 (1928).
^e"Pipe Friction Manual," 2d ed., Hydraulic Institute, New York, 1933.
^fSee, *J. Heat Eng.*, **62**, 131-143 (1940).
^gClawson, *Heat Transfer, Heat Piping Air Cond.*, 436, 443-447 (1952).
^hSnyder and Thomson, *Hydraulics*, 2d ed., McGraw-Hill, New York, 1934, p. 223.
ⁱHooper, *Indust. Chem. and Eng. Chem. Eng. Prog.*, **44**, 691-696 (1949).
^jGilman, *Heat Piping Air Cond.*, **27**(4), 141-147 (1955).
^kMcNown, *Proc. Am. Soc. Civ. Eng.*, **79**, Separate 259, 1-22 (1953); discussion, *ibid.*, **80**, Separate 326, 19-45 (1954). For the effect of branch spacing on friction losses in dividing flow, see Blacker, Neumann, and Quastler, *Proc. Am. Soc. Civ. Eng.*, *J. Hydraul. Div.*, **100**(1972), 205-229 (1977).
^lThis is pressure drop (including friction loss) between run and branch, based on velocity in the mainstem before branching. Actual value depends on the flow split, ranging from 0.5 to 1.5 of mainstem velocity run and from 0.7 to 1.5 of mainstem velocity branch.
^mLaminar Loss of Head in Flow of Fluids through Various Types of Fittings and Valves, *Trans. Inst. Chem. Eng.*, **34**, 140 (1947).

pressures are both atmospheric and the fluid velocities are 0 and $V = 2$ m/s, respectively, and there is no shaft work, simplify to

$$\rho gZ = \frac{\rho V^2}{2} + L$$

Contributing to L , are losses for the entrance to the pipe, the three sections of straight pipe, the butterfly valve, and the 90° bend. Note that no exit loss is used because the discharged jet is outside the control volume. Instead, the $V^2/2$ term accounts for the kinetic energy of the discharging stream. The Reynolds number in the pipe is

$$Re = \frac{\rho V D_p}{\mu} = \frac{998.25 \times 2 \times 0.025}{0.0001} = 1.00 \times 10^5$$

From Fig. 6-9 or Eq. (6-26), at $e/D = 0.046 \times 10^{-3}/0.025 = 0.00184$, the friction factor is about 0.0024. The straight pipe losses are then

$$L_{sp} = \left(\frac{4fL}{D} \right) \frac{V^2}{2} \\ = \left(\frac{4 \times 0.0024 \times (1 + 1 + 1)}{0.025} \right) \frac{V^2}{2} \\ = 1.23 \frac{V^2}{2}$$

The losses from Table 6-4 in terms of velocity heads K are $K = 0.5$ for the sudden contraction and $K = 0.52$ for the butterfly valve. For the 90° standard elbow ($r/D = 1$), the table gives $K = 0.75$. The method of Eq. (6-94), using Fig. 6-14, gives

$$K = K_1 C_1 + K_2 C_2 \\ = 0.24 + 1.24 \times 1.0 \times \left(\frac{0.0024}{0.0001} \right) \\ = 0.37$$

This value is more accurate than the value in Table 6-4. The value $f_{smooth} = 0.0044$ is obtainable either from Eq. (6-37) or Fig. 6-9.

The total losses are then

$$L = (1.23 + 0.5 + 0.52 + 0.37) \frac{V^2}{2} = 2.62 \frac{V^2}{2}$$

and the liquid level Z_2 is

$$Z = \frac{1}{g} \left(\frac{V^2}{2} + 2.62 \frac{V^2}{2} \right) = 1.62 \frac{V^2}{2g} \\ = \frac{1.62 \times 2^2}{2 \times 9.81} = 0.33 \text{ m}$$

Curved Pipes and Coils. For flow through curved pipe or coil, a secondary circulation perpendicular to the main flow called the **Dean effect** occurs. This circulation increases the friction relative to straight pipe flow and stabilizes laminar flow, delaying the transition Reynolds number to about

$$Re_{crit} = 2,100 \left(1 + 12 \sqrt{\frac{D_c}{D}} \right) \quad (6-100)$$

where D_c is the coil diameter. Equation (6-100) is valid for $10 < D_c/D < 250$. The **Dean number** is defined as

$$Dn = \frac{Re}{(D_c/D)^{3/2}} \quad (6-101)$$

In laminar flow, the friction factor for curved pipe f_c may be expressed in terms of the straight pipe friction factor $f = 16/Re$ as (Haert, *Chem. Eng. Sci.*, **43**, 775-782 (1958))

TABLE 6-5 Additional Frictional Loss for Laminar Flow through Fittings and Valves

Type of fitting or valve	Additional frictional loss expressed as K			
	$Re = 1,000$	500	100	50
90° ell, short radius	0.0	1.0	2.5	9.0
Gate valve	1.2	1.7	6.9	24
Globe valve, composition disk	1.1	1.2	20	39
Plug	1.2	1.4	19	27
Angle valve	0	6.5	11	19
Check valve, swing	0	0.5	17	35

Source: D. From curves by Kenesha and Bowley, *Trans. Am. Soc. Mech. Eng.*, **79**, 1706-1708 (1957).

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